AMENDMENTS TO THE SPECIFICATION:

Please add the following ***new*** paragraph on page 1, between lines 2 and 3:

CROSS-REFERENCE TO RELATED APPLICATIONS

This U.S. National stage application claims priority under 35 U.S.C. §119(a) to Japanese Patent Application No. 2004-144675, filed in Japan on May 14, 2004, the entire contents of which are hereby incorporated herein by reference.

Please replace paragraph [0002] beginning at page 1, line 9 with the following rewritten version:

As one of conventional rotary compressors including a compression mechanism in which a piston (an eccentric rotation body) rotates eccentrically within a cylinder chamber, there has been proposed a rotary compressor in which refrigerant is compressed by volume change of the cylinder chamber in association with eccentric rotation of an annular piston (for example, see ~~Patent Document 1~~Japanese Patent Publication No. 6-288358).

Please replace paragraph [0010] beginning at page 3, line 18 with the following rewritten version:

In the compressor (**100**) in this case, a support plate (**117**) for supporting the piston side end plate (**126B**) is formed at the lower face of the end plate (**126B**) connected to the annular piston (**122**). A sealing ring (**129**) is provided coaxially with the annular piston (**122**) at a part where the piston side end plate (**126B**) faces the support plate (**117**). The piston side end plate (**126B**) receives at a part thereof corresponding to the inner peripheral side of the sealing ring (**129**) pressure of the refrigerant in the high pressure space (**S**). This causes the piston side end plate (**126B**) to push upward in the axial direction towards the cylinder (**121**) to minimize gaps in the axial direction between the cylinder (**121**) and the annular piston (**123**) (a first axial-direction gap between the lower end face in the axial direction of the

cylinder (**121**) and the piston side end plate (**126B**) and a second axial-direction gap between the upper end face in the axial direction of the piston (**122**) and the cylinder side end plate (**126A**)).

~~Patent Document 1: Japanese Patent Application Laid Open Publication No. 6-288358A~~

Please remove the heading at page 4, line 4, as shown:

~~Disclosure of the Invention~~

Please remove the heading at page 4, line 5, as shown:

~~Problem that the Invention is to Solve~~

Please replace the heading at page 6, line 7, with the following rewritten version:

~~Means of Solving the Problems~~SUMMARY OF THE INVENTION

Please replace paragraph [0019] beginning at page 6, line 10 with the following rewritten version:

Specifically, ~~the~~ a first aspect of the present invention provides a rotary compressor includes: a compression mechanism (**20**) including a cylinder (**21**) having a cylinder chamber (**C**) (**C1**, **C2**), a piston (**22**) accommodated in the cylinder chamber (**C**) (**C1**, **C2**) eccentrically with respect to the cylinder (**21**), and a blade (**23**) arranged in the cylinder chamber (**C**) (**C1**, **C2**) and defining the cylinder chamber (**C**) (**C1**, **C2**) into a first chamber (**C-Hp**) (**C1-Hp**, **C2-Hp**) and a second chamber (**C-Lp**) (**C1-Lp**, **C2-Lp**), at least one of the cylinder (**21**) and the piston (**22**) rotating eccentrically as an eccentric rotation body (**21**, **22**); a drive shaft (**33**) for driving the compression mechanism (**20**); a pressing mechanism (**60**) for bringing a cylinder side end plate (**26A**), which is provided at one end in an axial direction of the cylinder chamber (**C**) (**C1**, **C2**) and faces an end face in an axial direction of the piston (**22**), and a piston side end plate (**26B**), which is provided at the other end in the axial

direction of the cylinder chamber (**C**) (**C1**, **C2**) and faces an end face in an axial direction of the cylinder (**21**), close to each other in an axial direction of the drive shaft (**33**); and a casing (**10**) for accommodating the compression mechanism (**20**), the drive shaft (**33**), and the pressing mechanism (**60**), wherein the pressing mechanism (**60**) is eccentric away from the center of the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**), and the pressing mechanism (**60**) generates axial-direction pressing force of which center is eccentric away from the center of the drive shaft (**33**). Wherein, "a part eccentric from the center of the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**) and eccentric from the center of the drive shaft (**33**)" is shortened to "a part eccentric from the center of the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**)" in the following description.

Please replace the paragraph [0020] beginning at page 7, line 5 with the following rewritten version:

In the first aspect of the present invention, the eccentric rotation body (**21**, **22**) eccentrically rotates by the drive shaft (**33**) to change each volume of the first chamber (**C-Hp**) (**C1-Hp**, **C2-Hp**) and the second chamber (**C-Lp**) (**C1-Lp**, **C2-Lp**) in the cylinder chamber (**C**) (**C1**, **C2**), resulting in compression of to-be-processed fluid. In the compression, the pressing mechanism (**60**) brings the piston side end plate (**26B**) and the cylinder side end plate (**26A**) close to each other in the axial direction to minimize gaps in the axial direction between the piston (**22**) and the cylinder (**21**) (a first axial-direction gap between the end face in the axial direction of the cylinder (**21**) and the piston side end plate (**26B**) and a second axial-direction gap between the end face in the axial direction of the piston (**22**) and the cylinder side end plate (**26A**)).

Please replace the paragraph [0021] beginning at page 7, line 15 with the following rewritten version:

In this first aspect of the present invention, the resultant force of the axial-direction pressing force obtained from the pressing mechanism (**60**) is centered at a part eccentric from the center of the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**). Thus,

separation in the axial direction of the point of action of the axial-direction pressing force (P) from the point of action of the thrust load (PT) is restrained, which is the difference from the aforementioned conventional technique. As a result, the turnover moment caused due to the thrust load (PT) can be restrained effectively.

Please replace the paragraph [0022] beginning at page 7, line 22 with the following rewritten version:

~~Referring to the~~ A second aspect of the present invention, ~~in~~ is the rotary compressor ~~of~~ according to the first aspect of the present invention, wherein the cylinder chamber (**C**) is in a circular shape in section at a right angle in an axial direction, and the piston (**22**) is formed of a circular piston (**22**) arranged in the cylinder chamber (**C**). Wherein, "the section at a right angle in the axial direction" herein means a section at a right angle with respect to the drive shaft (the rotation center).

Please replace the paragraph [0023] beginning at page 7, line 27 with the following rewritten version:

In the second aspect of the present invention, in the rotary compressor in which the cylinder chamber (**C**) has a circular shape in section at a right angle in the axial direction and the piston (**22**) is formed of a circular piston (**22**), the resultant force of the axial-direction pressing force obtained from the pressing mechanism (**60**) is centered at a part eccentric from the center of the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**), so that separation in the axial direction of the point of action of the axial-direction pressing force (P) from the point of action of the thrust load (PT) is restrained, restraining the turnover moment caused due to the thrust load (PT) effectively.

Please replace the paragraph [0024] beginning at page 8, line 8 with the following rewritten version:

~~Referring to the~~ A third aspect of the present invention, ~~in~~ is the rotary compressor of the first aspect of the present invention, wherein the cylinder chamber (**C1**, **C2**) is in an annular shape in section at a right angle in an axial direction, and the piston (**22**) is formed of an annular piston (**22**) arranged in the cylinder chamber (**C1**, **C2**) and defining the cylinder chamber (**C1**, **C2**) into an outer cylinder chamber (**C1**) and an inner cylinder chamber (**C2**).

Please replace the paragraph [0025] beginning at page 8, line 13 with the following rewritten version:

In the third aspect of the present invention, the annular piston (**22**) is arranged in the annular cylinder chamber (**C1, C2**) to form an outside cylinder chamber (the outer cylinder chamber) (**C1**) between the wall face on the outer peripheral side of the cylinder chamber (**C1**, **C2**) and the outer peripheral face of the annular piston (**22**) and an inside cylinder chamber (the inner cylinder chamber) (**C2**) between the wall face on the inner peripheral side of the cylinder chamber and the inner peripheral face of the annular piston (**22**). As a result, the rotary compressor can be attained in which the to-be-processed fluid is compressed by alternate repetition of volume expansion and contraction in both the outer cylinder chamber (**C1**) and the inner cylinder chamber (**C2**), similarly to the aforementioned conventional rotary compressor.

Please replace the paragraph [0026] beginning at page 8, line 23 with the following rewritten version:

In this third aspect of the present invention, similarly to the first and second aspects of the present inventions, the resultant force of the axial-direction pressing force obtained from the pressing mechanism (**60**) is centered at a part eccentric from the center of the end plate (**26A, 26B**) of the eccentric rotation body (**21, 22**), so that separation in the axial direction of the point of action of the axial-direction pressing force (P) from the point of action of the thrust load (PT) is restrained, resulting in effective restraint on the turnover moment caused due to the thrust load (PT).

Please replace the paragraph [0027] beginning at page 9, line 2 with the following rewritten version:

~~Referring to the~~ A fourth aspect of the present invention, ~~in~~ is the rotary compressor of the third aspect of the present invention, wherein the piston (**22**) is in a C-shape into which a part of an annular shape is divided, a swing bush (**27**) is provided so as to be slidably held at the divided part of the piston (**22**), a blade groove (**28**) being formed therein for holding a blade (**23**) so as to allow the blade (**23**) to move back and forth, and the blade (**23**) is inserted in the blade groove (**28**) so as to extend from a wall face on an inner peripheral side to a wall face on an outer peripheral side of the annular cylinder chamber (**C1**, **C2**).

Please replace the paragraph [0028] beginning at page 9, line 9 with the following rewritten version:

In the fourth aspect of the present invention, when at least one of the cylinder (**21**) and the piston (**22**) eccentrically rotates as the eccentric rotation body (**21**, **22**), the blade (**23**) moves back and forth with the face thereof being in face contact with the blade groove (**28**) in the swing bush (**27**) while the swing bush (**27**) rocks with the face thereof being in face contact with the divided part of the piston (**22**). Thus, the cylinder chambers (**C1**, **C2**) can be divided into first chambers (**C1-Hp**, **C2-Hp**) and second chambers (**C2-Lp**, **C2-Lp**) while the blade (**23**) moves smoothly in the eccentric rotation of the eccentric rotation body (**21**, **22**).

Please replace the paragraph [0029] beginning at page 9, line 16 with the following rewritten version:

~~Referring to the~~ A fifth aspect of the present invention, ~~in~~ is the rotary compressor of the first aspect of the present invention, wherein discharge ports (**45**, **46**) for discharging fluid compressed in the cylinder chamber (**C1**, **C2**) to outside of the compression mechanism (**20**) are formed in the compression mechanism (**20**), and the pressing mechanism (**60**) generates

the axial-direction pressing force of which center is eccentric to the discharge ports (**45**, **46**) away from the center of the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**).

Please replace the paragraph [0030] beginning at page 9, line 22 with the following rewritten version:

In the fifth aspect of the present invention, the to-be-processed fluid at high pressure by compression in, for example, the first chambers (**C1-Hp**, **C2-Hp**) is discharged outside the compression mechanism (**20**) through the discharge ports (**45**, **46**).

Please replace the paragraph [0031] beginning at page 9, line 25 with the following rewritten version:

In this fifth aspect of the present invention, the center of the resultant force of the axial-direction pressing force is set at a part near the discharge ports (**45**, **46**) in the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**) where the to-be-processed fluid is liable to be at high pressure and where the thrust load (PT) working on the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**) is liable to be large. Accordingly, the point of action of the axial-direction pressing force (P) readily agrees with the point of action of the thrust load (PT) in the axial direction, with a result that the turnover moment ~~cuased~~ caused due to the thrust load (PT) can be restrained further effectively.

Please replace the paragraph [0032] beginning at page 10, line 6 with the following rewritten version:

~~Referring to the~~ A sixth aspect of the present invention, ~~in~~ is the rotary compressor of the first aspect of the present invention, wherein a support plate (**17**) is arranged along a face opposite a face on the cylinder chamber (**C1**, **C2**) side of the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**) in the casing (**10**), a sealing ring (**29**) for defining an opposing part (**61**, **62**) between the end plate (**26A**, **26B**) and the support plate (**17**) inside and outside in a radial direction into a first opposing section (**61**) and a second opposing section

(**62**) is arranged eccentrically away from the center of the eccentric rotation body (**21**, **22**) in one of the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**) and the support plate (**17**), and the pressing mechanism (**60**) allows pressure of fluid discharged outside the compression mechanism (**20**) to work on the first opposing section (**61**) in the end plate (**26A**, **26B**).

Please replace the paragraph [0033] beginning at page 10, line 16 with the following rewritten version:

In the sixth aspect of the present invention, the sealing ring (**29**) is provided between the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**) and the support plate (**17**) to partition an opposing part between the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**) and the support plate (**17**) into two or more opposing sections (**61**, **62**). The fluid at high pressure in the compression mechanism (**20**) is introduced into the first opposing section (**61**) and the pressure of the fluid is allowed to work on the first opposing section (**61**) in the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**), thereby obtaining the axial-direction pressing force against the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**).

Please replace the paragraph [0034] beginning at page 10, line 25 with the following rewritten version:

In the sixth aspect of the present invention, the sealing ring (**29**) is provided at a part eccentric from the center of the eccentric rotation body (**21**, **22**), so that the axial-direction pressing force obtained from the sealing ring (**29**) is centered at a part eccentric from the center of the end plate (**26A**, **26B**) of the eccentric rotation body (**21**, **22**). This restrains separation of the point of action of the axial-direction pressing force (P) from the point of action of the thrust load (PT), as described above.

Please replace the paragraph [0035] beginning at page 11, line 4 with the following rewritten version:

~~Referring to the~~ A seventh aspect of the present invention, ~~in~~ is the rotary compressor of the sixth aspect of the present invention, wherein the sealing ring (**29**) is fitted in an annular groove (**17b**) formed in one of the eccentric rotation body (**21, 22**) and the support plate (**17**).

Please replace the paragraph [0036] beginning at page 11, line 7 with the following rewritten version:

In the seventh aspect of the present invention, the sealing ring (**29**) is fitted in the annular groove (**17b**), thereby being held securely at a position eccentric from the center of the eccentric rotation body (**21, 22**).

Please replace the paragraph [0037] beginning at page 11, line 10 with the following rewritten version:

~~Referring to the~~ An eighth aspect of the present invention, ~~in~~ is the rotary compressor of the first aspect of the present invention, wherein a slit (**63**) is formed at a part eccentric away from the center of the eccentric rotation body (**21**) in a face portion opposite a face on the cylinder chamber (**C1, C2**) side of the end plate (**26A**) of the eccentric rotation body (**21**), and the pressing mechanism (**60**) allows pressure of fluid discharged outside the compression mechanism (**20**) to work on the slit (**63**).

Please replace the paragraph [0038] beginning at page 11, line 16 with the following rewritten version:

In the ~~eight~~ eighth aspect of the present invention, the pressure of the fluid at high pressure in the compression mechanism (**20**) is allowed to work on the slit (**63**) to cause the axial-direction pressing force (P) to readily work in the vicinity of the slit (**63**) in the end plate (**26A**) of the eccentric rotation body (**21**). In this aspect of the present invention, the slit (**63**) to be formed at a part eccentric from the center of the eccentric rotation body (**21**). This allows the axial-direction pressing force obtained according to the shape of the slit (**63**) is

centered at a part of the end plate (**26A**) eccentric from the center of the eccentric rotation body (**21**). Accordingly, separation of the point of action of the axial-direction pressing force (P) from the point of action of the thrust load (PT) in the axial direction is restrained.

Please replace the paragraph [0039] beginning at page 11, line 25 with the following rewritten version:

~~Referring to the~~ A ninth aspect of the present invention, ~~in~~ is the rotary compressor of the first aspect of the present invention, wherein a groove (**65**) and a through hole (**64**) are formed, the groove (**65**) being formed in a portion eccentric away from the center of the eccentric rotation body (**21**) on a face opposite a face on the cylinder chamber (**C1**, **C2**) side of the end plate (**26A**) of the eccentric rotation body (**21**) and the through hole (**64**) being formed in the end plate (**26A**) for allowing the groove (**65**) to communicate with the cylinder chamber (**C**) (**C1**, **C2**), and the pressing mechanism (**60**) introduces part of fluid compressed in the cylinder chamber (**C1**, **C2**) into the groove (**65**) through the through hole (**64**) to allow the pressure of the fluid to work on the groove (**65**).

Please replace the paragraph [0040] beginning at page 12, line 8 with the following rewritten version:

In the ninth aspect of the present invention, part of the fluid compressed in the compression mechanism (**20**) is introduced into the groove (**65**) through the through hole (**64**), so that the axial-direction pressing force readily works in the vicinity of the groove (**65**) in the end plate (**26A**) of the eccentric rotation body (**21**). In this invention, the groove (**65**) is formed in a part eccentric from the center of eccentric rotation body (**21**). This allows the axial-direction pressing force obtained according to the shape of the grove (**65**) to be centered at a part of the end plate (**26A**) eccentric from the center of the eccentric rotation body (**21**). Accordingly, separation of the point of action of the axial-direction pressing force (P) from the point of action of the thrust load (PT) in the axial direction is restrained.

Please replace the paragraph [0041] beginning at page 12, line 16 with the following rewritten version:

~~Referring to the~~ A tenth aspect of the present invention, is the rotary compressor of the first aspect of the present invention further ~~includes:~~ including a sealing mechanism (**71**, **72**, **73**) for preventing leakage of fluid in at least one of a first axial direction gap between an end face in the axial direction of the cylinder (**21**) and the piston side end plate (**26B**) and a second axial direction gap between an end face in the axial direction of the piston (**22**) and the cylinder side end plate (**26A**).

Please replace the paragraph [0042] beginning at page 12, line 21 with the following rewritten version:

In the tenth aspect of the present invention, the sealing mechanism for minimizing the axial-direction gaps between the cylinder (**21**) and the piston (**22**) is provided in addition to the aforementioned pressing mechanism (**60**), so that the fluid at high pressure in, for example, the first chambers (**C1-Hp**, **C2-Hp**) is prevented from leaking into the second chambers (**C1-Lp**, **C2-Lp**) through the axial-direction gaps in the eccentric rotation of the eccentric rotation body (**21**, **22**).

Please replace the paragraph [0043] beginning at page 12, line 27 with the following rewritten version:

~~Referring to the~~ An eleventh aspect of the present invention, ~~in~~ is the rotary compressor of the tenth aspect of the present invention, the sealing mechanism is a chip seal (**71**, **72**, **73**) provided at least one of the first axial direction gap and the second axial direction gap.

Please replace the paragraph [0044] beginning at page 13, line 3 with the following rewritten version:

In the tenth aspect of the present invention, the chip seal (**71**, **72**, **73**) is provided at at least one of the first axial-direction gap and the second axial-direction gap between the cylinder (**21**) and the piston (**22**), minimizing the axial-direction gaps to prevent the fluid in the gaps form leaking.

Please replace the paragraph [0045] beginning at page 13, line 8 with the following rewritten version:

According to the first aspect of the present invention, in the compression mechanism (**20**) including the cylinder (**21**) having the cylinder chamber (**C1**) (**C1**, **C2**) and the piston (**22**), the pressing mechanism (**60**) minimizes the axial-direction gaps between the piston (**22**) and the cylinder (**21**), and the eccentric rotation body (**21**, **22**) eccentrically rotates to allow the axial-direction pressing force (P) to work against the thrust load (PT) caused in the cylinder chamber (**C**) (**C1**, **C2**). Working of the axial-direction pressing force (P) on the end plate (**26A**, **26B**) with the center thereof being eccentric from the center of the eccentric rotation body (**21**, **22**) minimizes separation of the axial-direction pressing force (P) from the thrust load (PT) in the radial direction, thereby restraining the turnover moment effectively.

Please replace the paragraph [0046] beginning at page 13, line 18 with the following rewritten version:

According to the second aspect of the present invention, in the compression mechanism (**20**) including the cylinder (**21**) having the circular cylinder chamber (**C1**) and the circular piston (**22**), the pressing mechanism (**60**) minimizes the axial-direction gaps between the piston (**22**) and the cylinder (**21**), and the eccentric rotation body (**21**, **22**) eccentrically rotates to allow the axial-direction pressing force (P) to work against the thrust load (PT) caused in the cylinder chamber (**C1**). Working of the axial-direction pressing force (P) on the end plate (**26A**, **26B**) with the center thereof being eccentric from the center of the eccentric rotation body (**21**, **22**) minimizes separation of the axial-direction pressing force (P) from the thrust load (PT) in the radial direction, thereby restraining the turnover moment effectively.

Please replace the paragraph [0047] beginning at page 13, line 27 with the following rewritten version:

According to the third aspect of the present invention, in the compression mechanism (**20**) including the cylinder (**21**) having the annular cylinder chamber (**C1**, **C2**) and the annular piston (**22**), the pressing mechanism (**60**) minimizes the axial-direction gaps between the piston (**22**) and the cylinder (**21**), and the eccentric rotation body (**21**, **22**) eccentrically rotates to allow the axial-direction pressing force (P) to work against the thrust load (PT) caused in the cylinder chamber (**C1**, **C2**). Working of the axial-direction pressing force (P) on the end plate (**26A**, **26B**) with the center thereof being eccentric from the center of the eccentric rotation body (**21**, **22**) minimizes separation of the axial-direction pressing force (P) from the thrust load (PT) in the radial direction, thereby restraining the turnover moment effectively.

Please replace the paragraph [0048] beginning at page 14, line 10 with the following rewritten version:

According to the fourth aspect of the present invention, in the rotary compressor of the third aspect of the present invention, the blade (**23**) moves back and forth with the face thereof being in face contact with the blade groove (**28**) in the swing bush (**27**) while the swing bush (**27**) rocks at the divided part of the piston (**22**), enabling the eccentric rotation body (**21**, **22**) to be in smooth eccentric rotation with the cylinder chamber (**C1**, **C2**) divided. Hence, seizing and abrasion at the contact part between the blade (**23**) and the swing bush (**27**) can be prevented and gas is prevented from leaking between the first chamber (**C1-Hp**, **C2-Hp**) and the second chamber (**C2-Lp**, **C2-Lp**).

Please replace the paragraph [0049] beginning at page 14, line 18 with the following rewritten version:

In the fifth aspect of the present invention, the axial-direction pressing force (P) against the end plate (**26A, 26B**) obtained from the pressing mechanism (**60**) is allowed to work on a part near the discharge ports (**45, 46**), which is liable to receive the thrust load (PT) in the cylinder chamber (**C1, C2**). Accordingly, the point of action of the axial-direction pressing force (P) can be brought close to the point of action of the thrust load (PT), reducing the turnover moment further effectively.

Please replace the paragraph [0050] beginning at page 14, line 24 with the following rewritten version:

According to the sixth aspect of the present invention, the pressing mechanism (**60**) is so composed that the pressure of the fluid at high pressure is allowed to work on the first opposing section (**61**) into which the end plate (**26A, 26B**) is defined by the sealing ring (**69**). The pressing mechanism (**60**) is easily composed by arranging the sealing ring (**69**) eccentrically from the center of the eccentric rotation body (**21, 22**), attaining effective reduction in turnover moment. Thus, the effect of reducing the turnover moment can be obtained with the simple construction.

Please replace the paragraph [0052] beginning at page 15, line 7 with the following rewritten version:

According to the seventh aspect of the present invention, the annular groove (**17b**) is formed in the piston (**22**) or the support plate (**17**), so that the sealing ring (**29**) can be held securely at a predetermined position.

Please replace the paragraph [0053] beginning at page 15, line 10 with the following rewritten version:

According to the ~~eight~~ eighth aspect of the present invention, the pressing mechanism (**60**) is so composed that the pressure of the fluid at high pressure is allowed to work on the slit (**63**) formed in the end plate (**26A**). The pressing mechanism (**60**) is easily composed by forming the slit (**63**) eccentrically from the center of the eccentric rotation body (**21**),

attaining effective reduction in turnover moment. Thus, the effect of reducing the turnover moment can be obtained with the simple construction.

Please replace the paragraph [0055] beginning at page 15, line 19 with the following rewritten version:

According to the ninth aspect of the present invention, the pressing mechanism (**60**) is so composed that part of the fluid compressed in the cylinder chamber (**C1**, **C2**) is allowed to work on the groove (**65**) through the through hole (**64**). The pressing mechanism (**60**) can be easily composed by forming the groove (**65**) eccentrically from the center of the eccentric rotation body (**21**), attaining effective reduction in turnover moment.

Please replace the paragraph [0056] beginning at page 15, line 24 with the following rewritten version:

Further, according to this ninth aspect of the present invention, as the pressure in the cylinder chamber (**C1**, **C2**) rises and the thrust load (PT) becomes large, the axial-direction pressing force (P) working on the groove (**65**) increases. In contrast, when the thrust load (PT) becomes small, the axial-direction pressing force (P) decreases. Hence, an increase in mechanical loss of the eccentric rotation body (**21**), which is caused due to surplus axial-direction pressing force (P), is prevented, implementing effective reduction in turnover moment.

Please replace the paragraph [0057] beginning at page 16, line 3 with the following rewritten version:

According to the tenth aspect of the present invention and the eleventh aspect of the present invention, the sealing mechanism (**71**, **72**, **73**) is provided in addition to the pressing mechanism (**60**), so that the fluid is prevented from leaking in the axial-direction gaps between the cylinder (**21**) and the piston (**22**), further increasing the compression efficiency.

Please remove the heading at page 17, line 8, as shown:

~~Description of Reference Numerals~~

Please remove paragraph [0059] at page 17, line 9, as shown:

~~**1** compressor~~

~~**10** casing~~

~~**17** lower housing (support plate)~~

~~**20** compression mechanism~~

~~**21** cylinder~~

~~**22** piston~~

~~**23** blade~~

~~**26A** cylinder side end plate~~

~~**26B** piston side end plate~~

~~**27** swing bush~~

~~**29** sealing ring~~

~~**33** drive shaft~~

~~**C1** cylinder chamber (outer cylinder chamber)~~

~~**C2** cylinder chamber (inner cylinder chamber)~~

~~**C1-Hp** first chamber (high pressure chamber)~~

~~**C2-Hp** first chamber (high pressure chamber)~~

~~**C1-Lp** second chamber (low pressure chamber)~~

~~**C2-Lp** second chamber (low pressure chamber)~~

~~**45, 46** discharge port~~

~~**60** pressing mechanism~~

~~**61** first opposing section~~

~~**71** chip seal~~

~~**72** chip seal~~

~~**73** chip seal~~

Please replace the heading at page 18, line 6, with the following rewritten version:

~~Best Mode for Carrying out~~ Detailed Description of the Invention